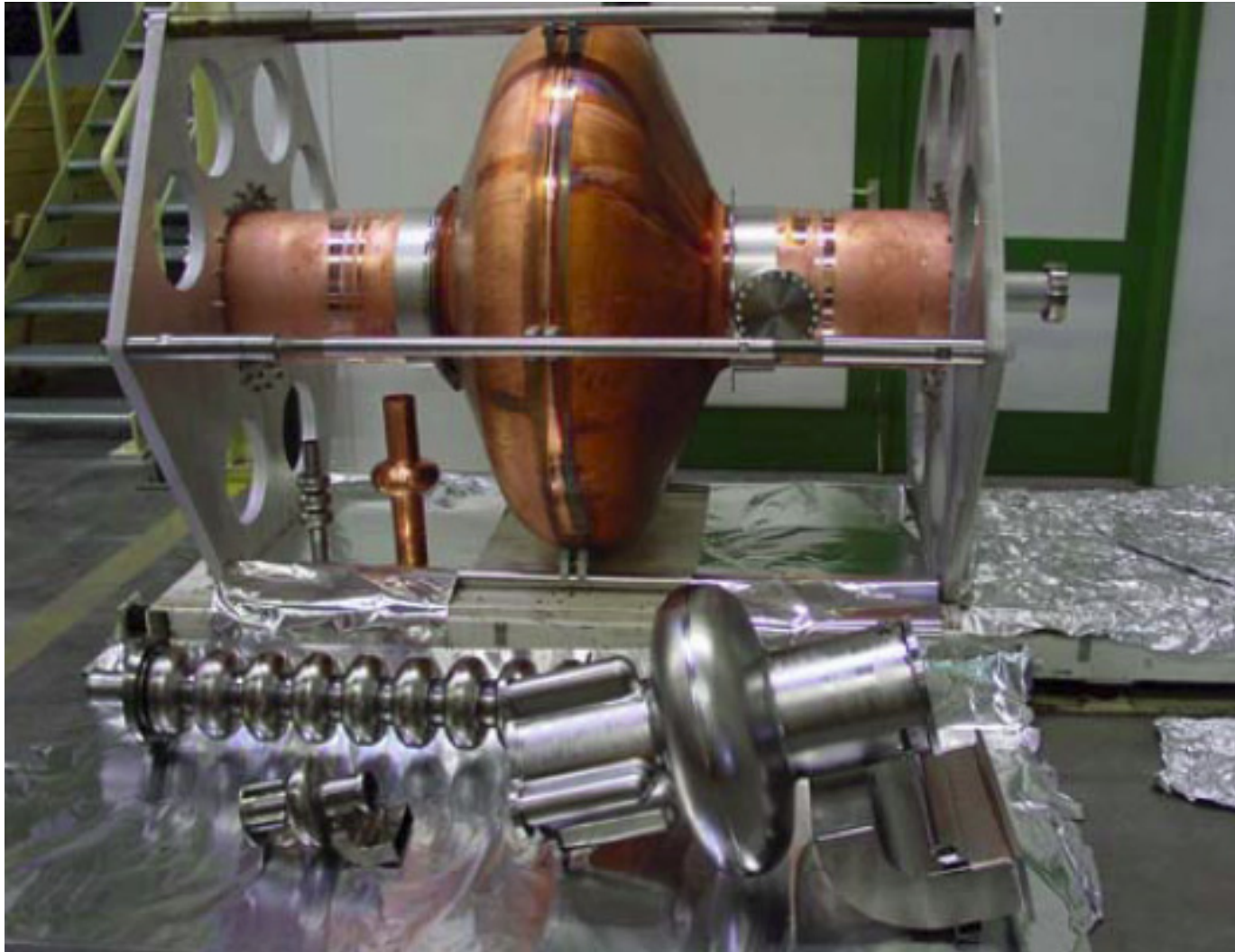


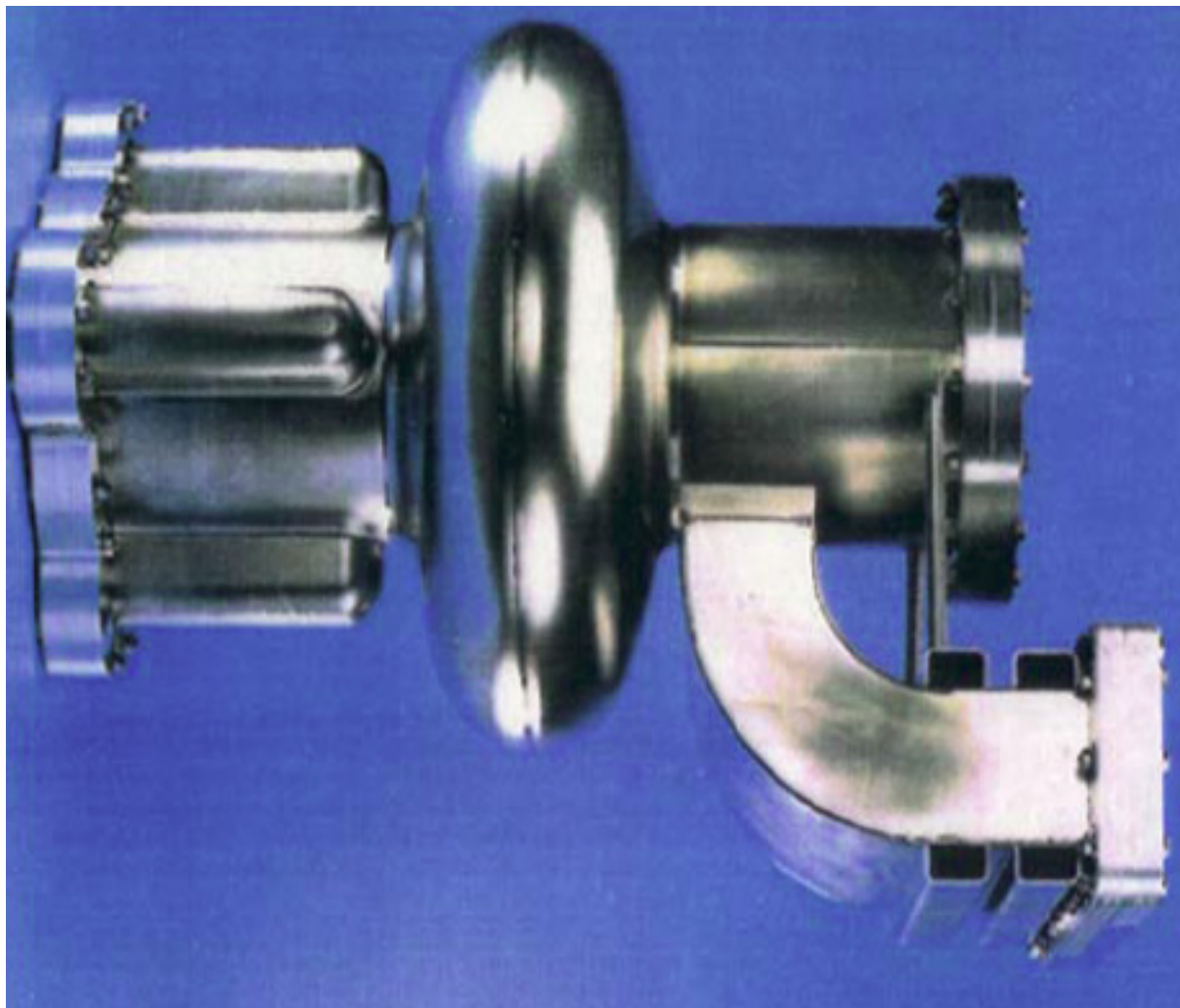
TM CLASS CAVITY DESIGN

Jean Delayen

Thomas Jefferson National Accelerator Facility
Old Dominion University



500 MHz, Single-cell



350 MHz, 4-cell, Nb on Cu



1500 MHz, 5-cell



1300 MHz 9-cell



Pill Box Cavity

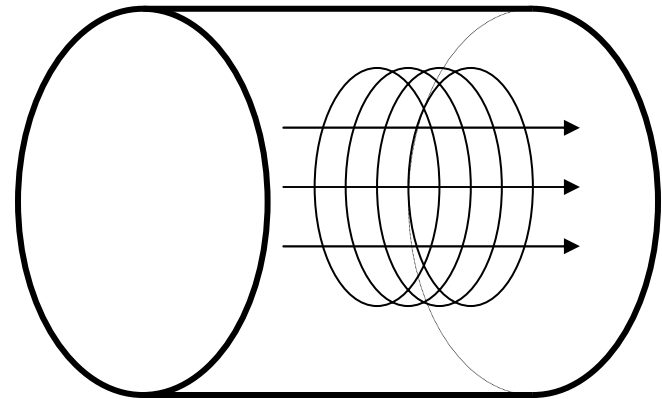
Hollow right cylindrical enclosure

Operated in the TM_{010} mode

$$H_z = 0$$

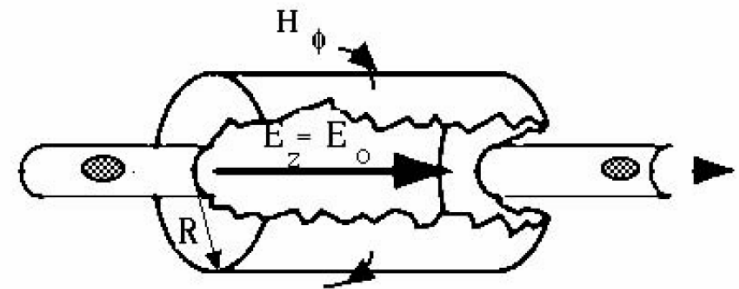
$$\frac{\partial^2 E_z}{\partial^2 r} + \frac{1}{r} \frac{\partial E_z}{\partial r} = \frac{1}{c^2} \frac{\partial^2 E_z}{\partial^2 t} \quad \omega_0 = \frac{2.405c}{R}$$

TM_{010} mode



$$E_z(r, z, t) = E_0 J_0 \left(2.405 \frac{r}{R} \right) e^{-i\omega_0 t}$$

$$H_\phi(r, z, t) = -i \frac{E_0}{\mu_0 c} J_1 \left(2.405 \frac{r}{R} \right) e^{-i\omega_0 t}$$



Modes in Pill Box Cavity

- TM_{010}
 - Electric field is purely longitudinal
 - Electric and magnetic fields have no angular dependence
 - Frequency depends only on radius, independent on length
- TM_{0mn}
 - Monopoles modes that can couple to the beam and exchange energy
- TM_{1mn}
 - Dipole modes that can deflect the beam
- TE modes
 - No longitudinal E field
 - Cannot couple to the beam

TM_{lmn} Modes in a Pill Box Cavity

$$\frac{E_r}{E_0} = -\frac{n\pi R}{x_{lm} L} J_l' \left(x_{lm} \frac{r}{R} \right) \sin \left(n\pi \frac{z}{L} \right) \cos l\varphi$$

$$\frac{E_\varphi}{E_0} = \frac{ln\pi R^2}{x_{lm}^2 rL} J_l \left(x_{lm} \frac{r}{R} \right) \sin \left(n\pi \frac{z}{L} \right) \sin l\varphi$$

$$\frac{E_z}{E_0} = J_l \left(x_{lm} \frac{r}{R} \right) \sin \left(n\pi \frac{z}{L} \right) \cos l\varphi$$

$$\omega_{lmn} = c \sqrt{\left(\frac{x_{lm}}{R} \right)^2 + \left(\frac{\pi n}{L} \right)^2}$$

$$\frac{H_r}{E_0} = -i\omega\epsilon \frac{l}{x_{lm}^2} \frac{R^2}{r} J_l \left(x_{lm} \frac{r}{R} \right) \cos \left(n\pi \frac{z}{L} \right) \sin l\varphi$$

x_{lm} is the m th root of $J_l(x)$

$$\frac{H_\varphi}{E_0} = -i\omega\epsilon \frac{R}{x_{lm}} J_l' \left(x_{lm} \frac{r}{R} \right) \cos \left(n\pi \frac{z}{L} \right) \cos l\varphi$$

$$\frac{H_z}{E_0} = 0$$

TM₀₁₀ Mode in a Pill Box Cavity

$$E_r = E_\phi = 0 \qquad E_z = E_0 J_0 \left(x_{01} \frac{r}{R} \right)$$
$$H_r = H_z = 0 \qquad H_\phi = -i\omega\epsilon E_0 \frac{R}{x_{01}} J_1 \left(x_{01} \frac{r}{R} \right)$$

$$\omega = x_{01} \frac{c}{R} \qquad x_{01} = 2.405$$

$$R = \frac{x_{01}}{2\pi} \lambda = 0.383 \lambda$$

TM₀₁₀ Mode in a Pill Box Cavity

Energy content

$$U = \epsilon_0 E_0^2 \frac{\pi}{2} J_1^2(x_{01}) LR^2$$

Power dissipation

$$P = E_0^2 \frac{R_s}{\eta^2} \pi J_1^2(x_{01}) (R + L) R$$

$$x_{01} = 2.40483$$

$$J_1(x_{01}) = 0.51915$$

Geometrical factor

$$G = \eta \frac{x_{01}}{2} \frac{L}{(R + L)}$$

TM010 Mode in a Pill Box Cavity

Energy Gain

$$\Delta W = E_0 \frac{\lambda}{\pi} \sin \frac{\pi L}{\lambda}$$

Gradient

$$E_{acc} = \frac{\Delta W}{\lambda/2} = E_0 \frac{2}{\pi} \sin \frac{\pi L}{\lambda}$$

Shunt impedance

$$R_{sh} = \frac{\eta^2}{R_s} \frac{1}{\pi^3 J_1^2(x_{01})} \frac{\lambda^2}{R(R+L)} \sin^2 \left(\frac{\pi L}{\lambda} \right)$$

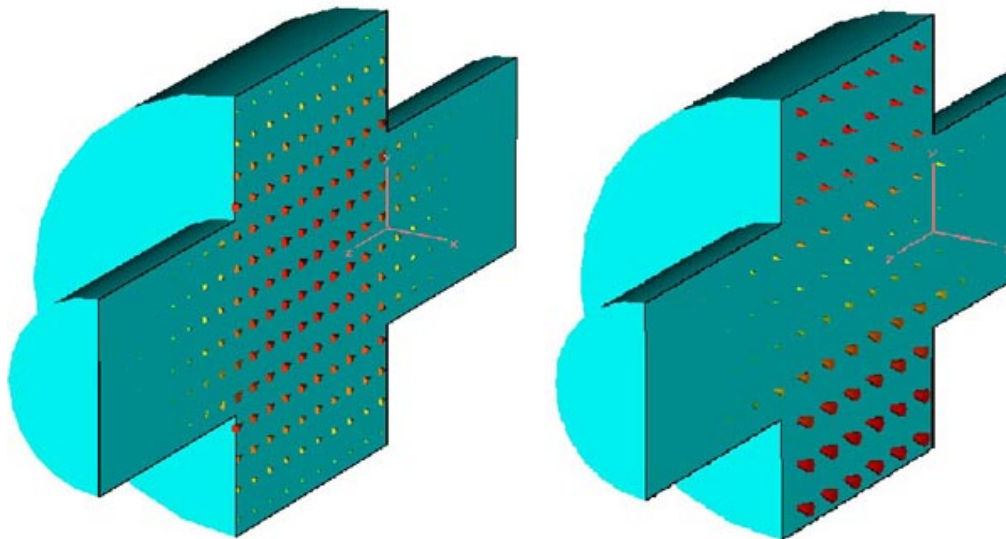
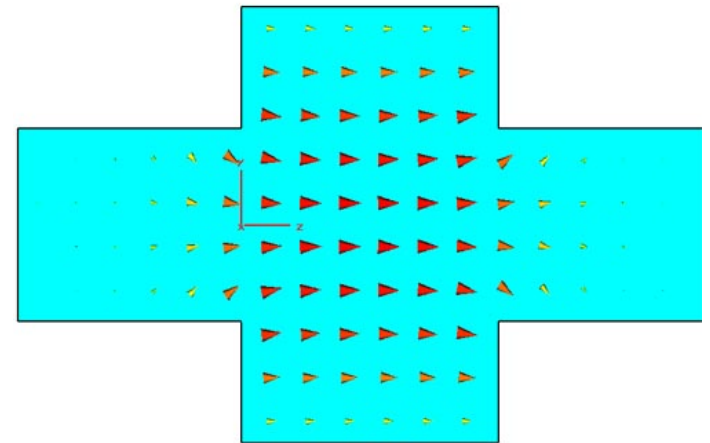
Real Cavities

Beam tubes reduce the electric field on axis

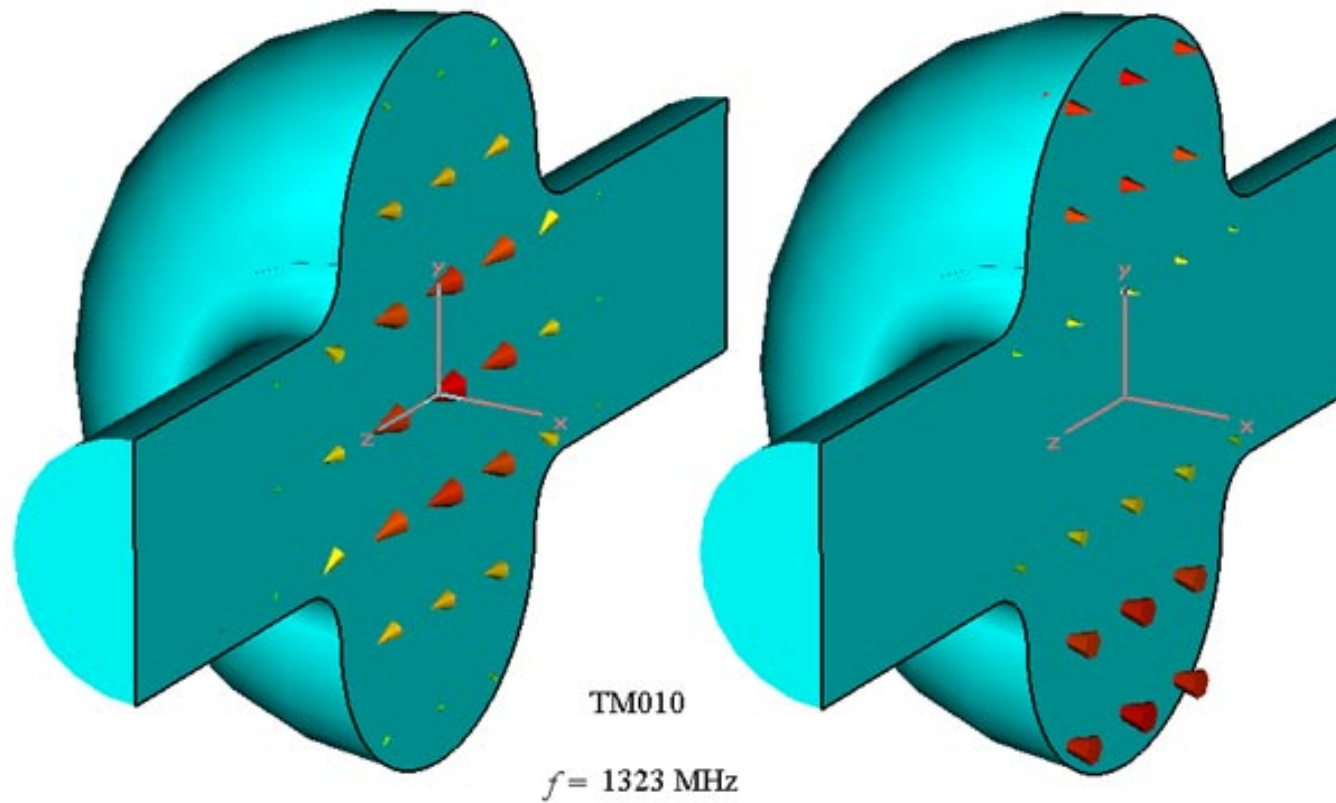
Gradient decreases

Peak fields increase

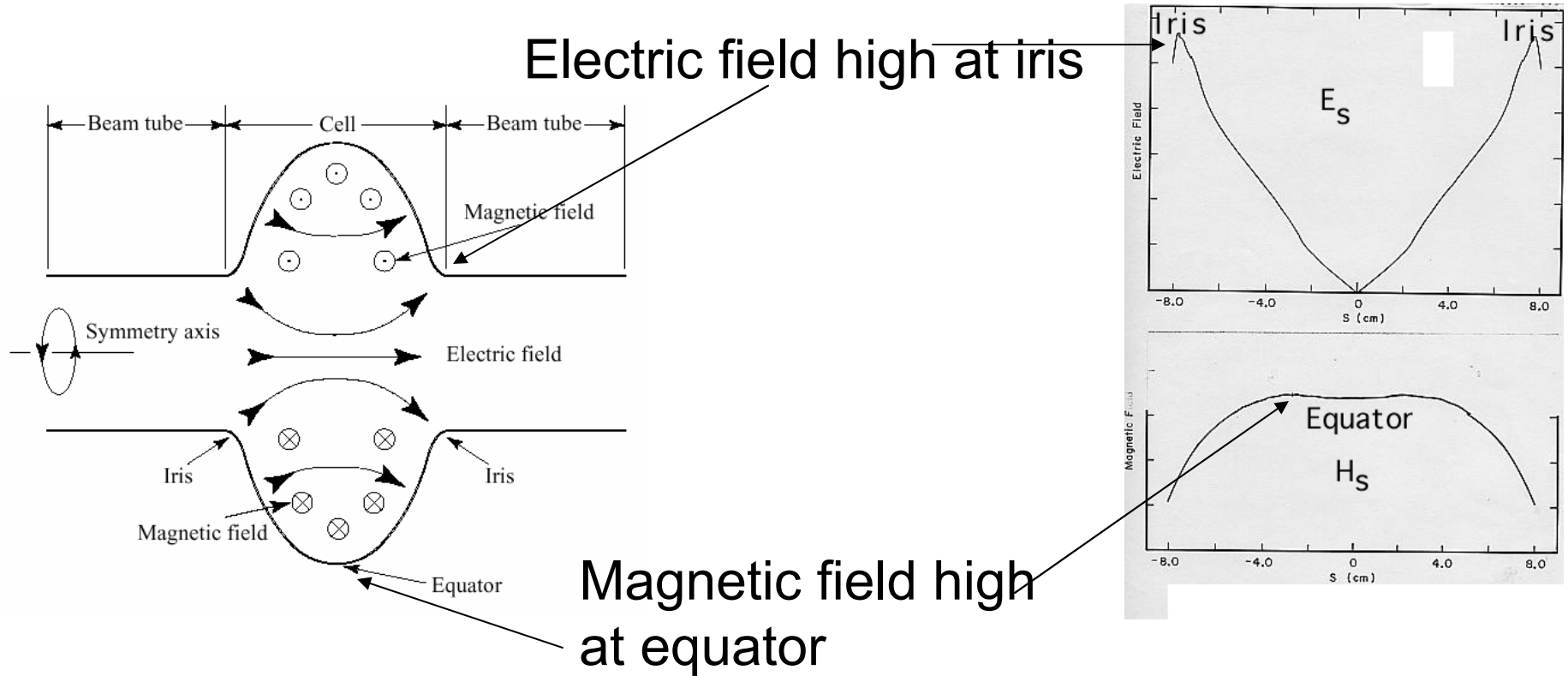
R/Q decreases



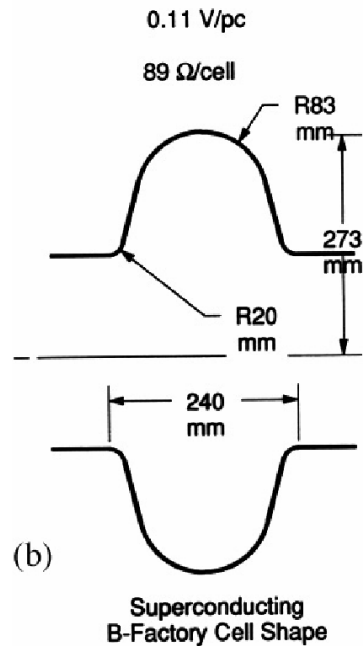
Real Cavities



Single Cell Cavities

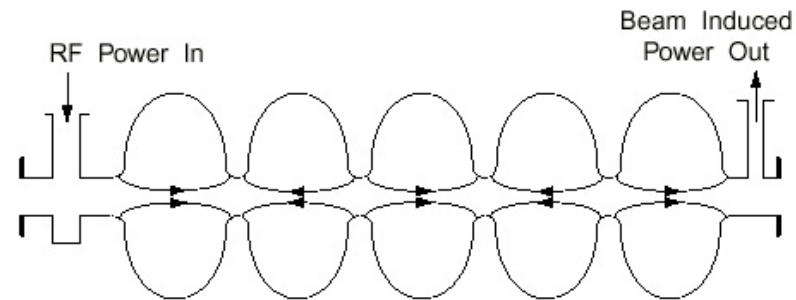


Single Cell Cavities

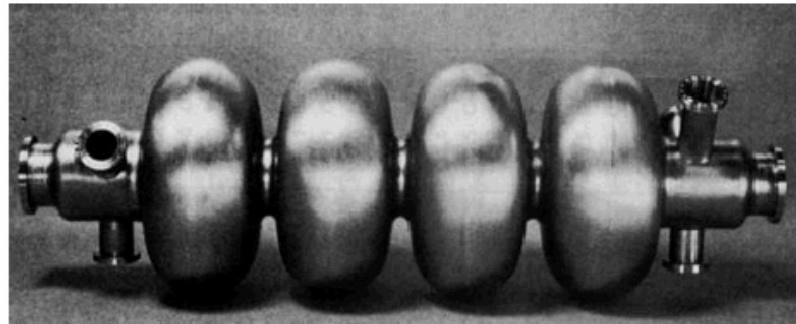


Quantity	Cornell SC 500 MHz	Pillbox
G	270 ohm Ω	257 Ω
R_a/Q_0	88 ohm/cell	196 Ω /cell
E_{pk}/E_{acc}	2.5	1.6
H_{pk}/E_{acc}	52 Oe/MV/m	30.5 Oe/(MV/m)

Multi-Cell Cavities

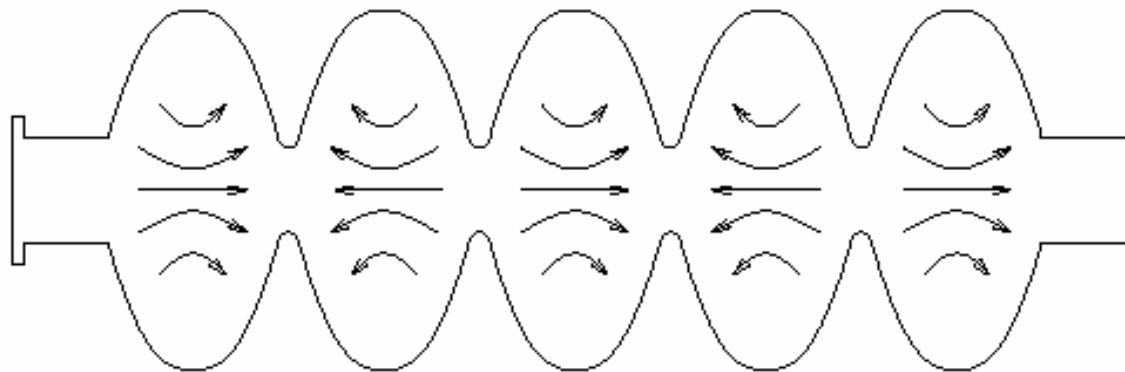
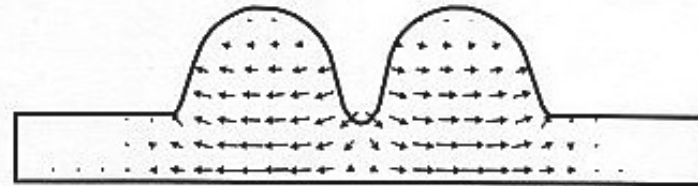
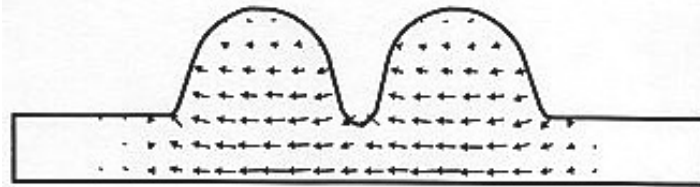


(c)



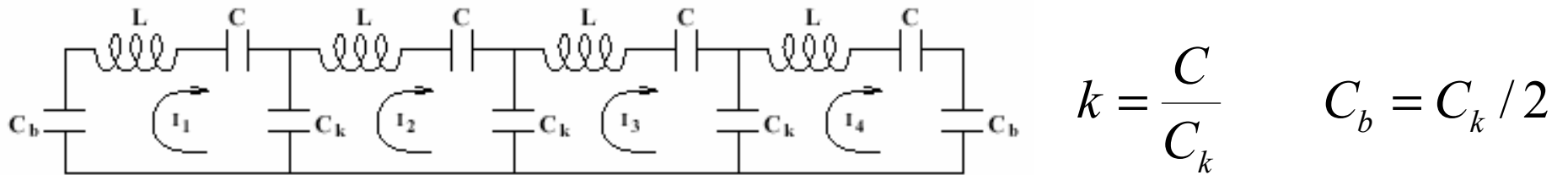
Multi-Cell Cavities

Modes of a 2 Cell Cavity



: Sketch of the electric field lines of the π -mode of a 5-cell :

Multi-Cell Cavities



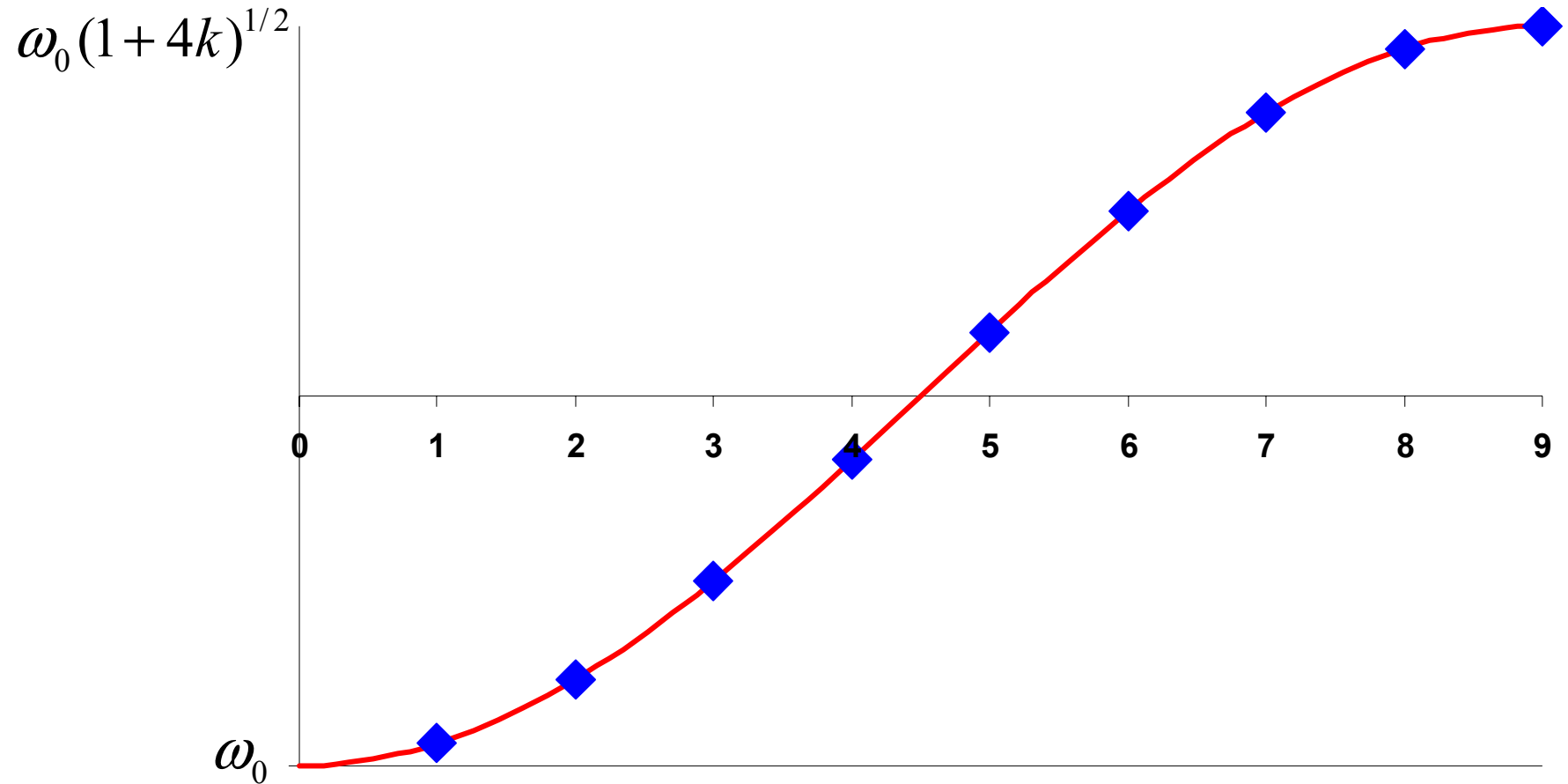
Mode frequencies: $\frac{\omega_m^2}{\omega_0^2} = 1 + 2k \left(1 - \cos \frac{\pi m}{n} \right)$

$$\frac{\omega_n - \omega_{n-1}}{\omega_0} \approx k \left(1 - \cos \frac{\pi}{n} \right) \approx \frac{k}{2} \left(\frac{\pi}{n} \right)^2$$

Voltages in cells: $V_j^m = \sin \left(\pi m \frac{2j-1}{2n} \right)$

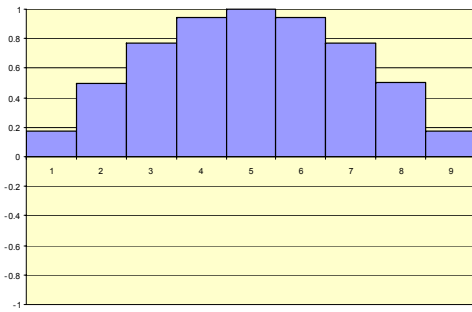
Pass-Band Modes Frequencies

9-cell cavity

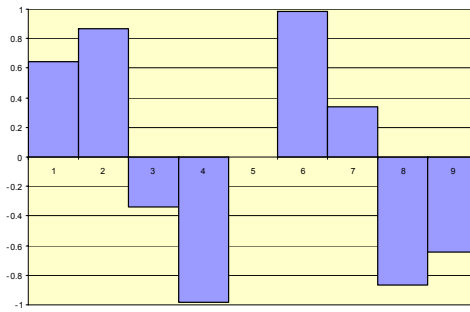


Cell Excitations in Pass-Band Modes

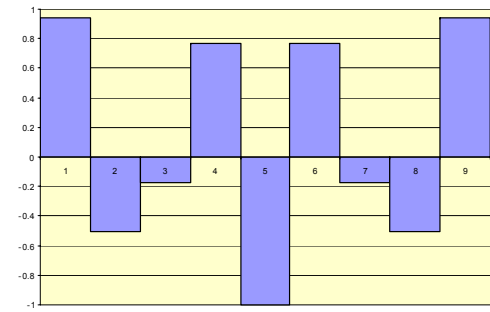
9 Cell, Mode 1



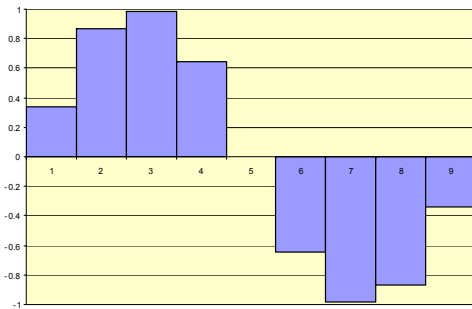
9 Cell, Mode 4



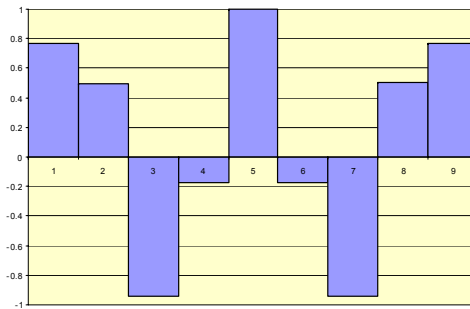
9 Cell, Mode 7



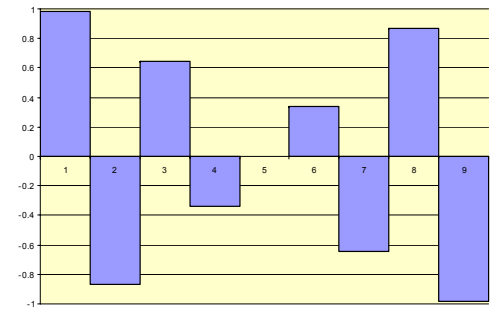
9 Cell, Mode 2



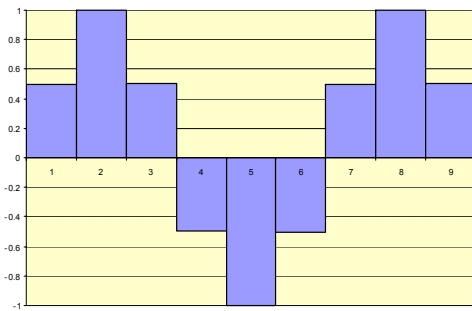
9 Cell, Mode 5



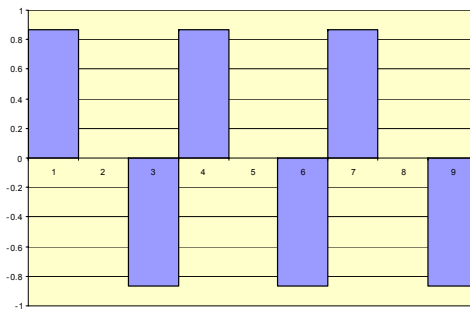
9 Cell, Mode 8



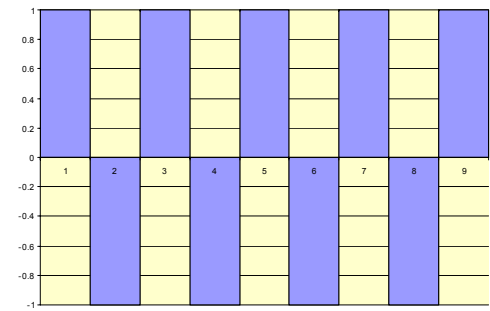
9 Cell, Mode 3



9 Cell, Mode 6



9 Cell, Mode 9

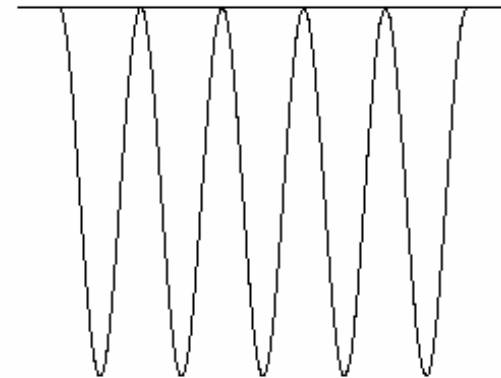
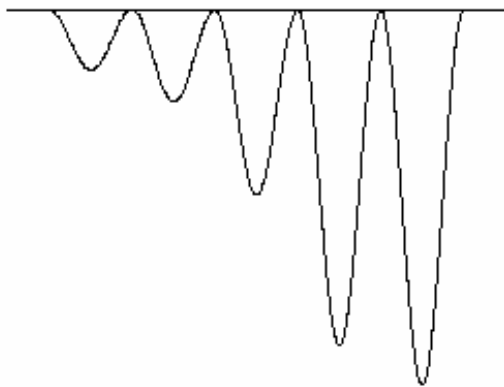


Field Flatness

Geometrical differences between cells causes a mixing of the eigenmodes

Sensitivity to mechanical deformation depends on mode spacing

$$\frac{\omega_n - \omega_{n-1}}{\omega_0} \simeq k \left(1 - \cos \frac{\pi}{n} \right) \simeq \frac{k}{2} \left(\frac{\pi}{n} \right)^2$$

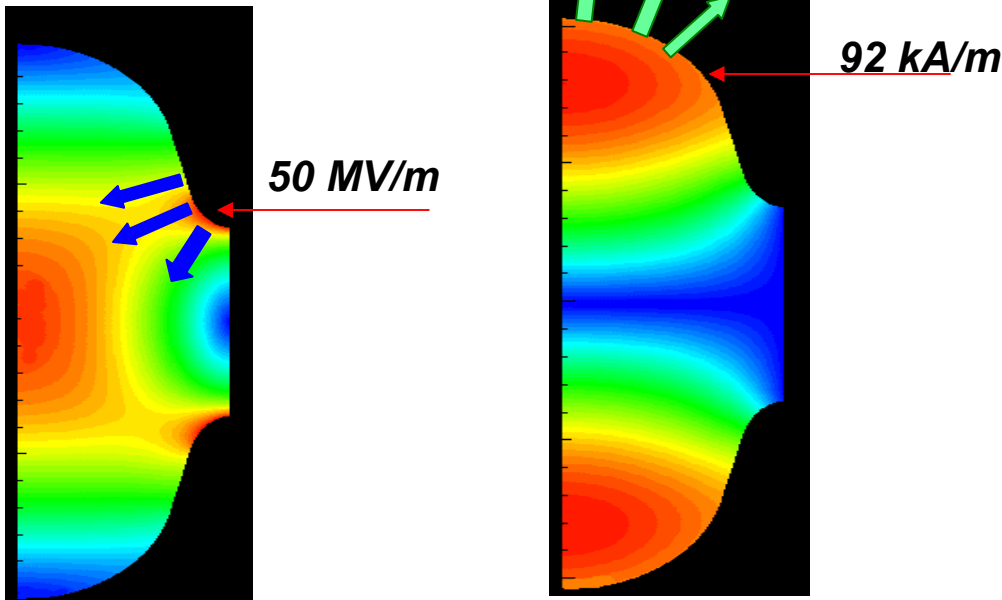


Mechanical Design

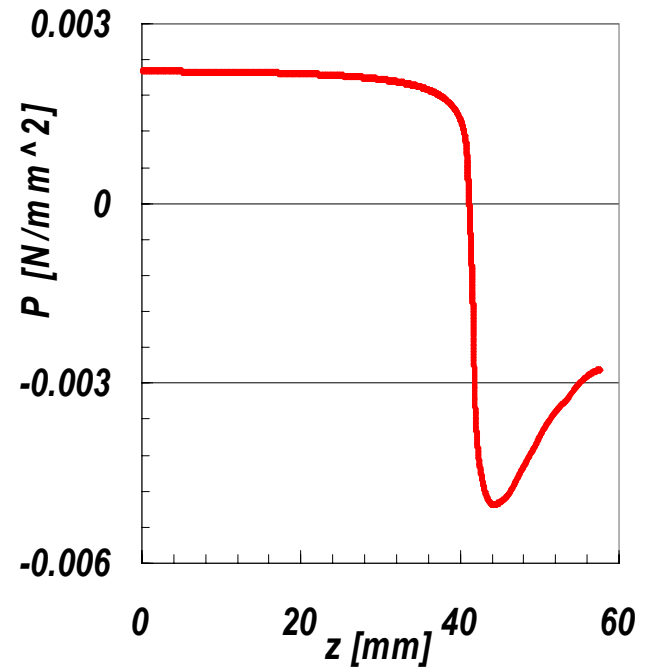
The mechanical design of a cavity follows its RF design:

- Lorentz Force Detuning
- Mechanical Resonances

Lorentz Force Detuning

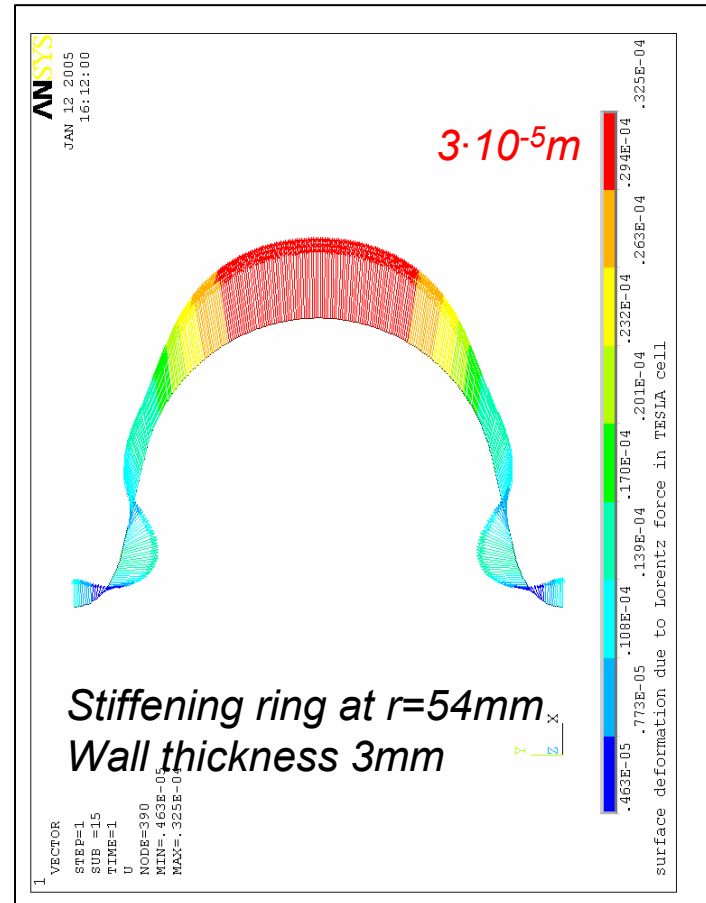
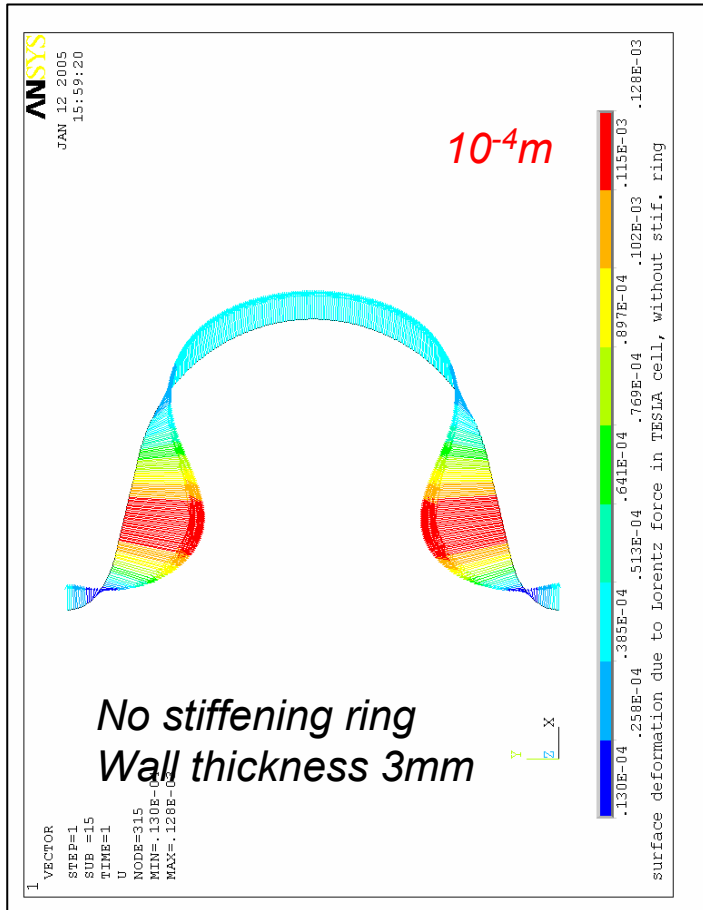


$$P = \frac{\mu_0 H_s^2 - \epsilon_0 E_s^2}{4}$$



E and H at $E_{acc} = 25$ MV/m in TESLA inner-cup

Mechanical Design



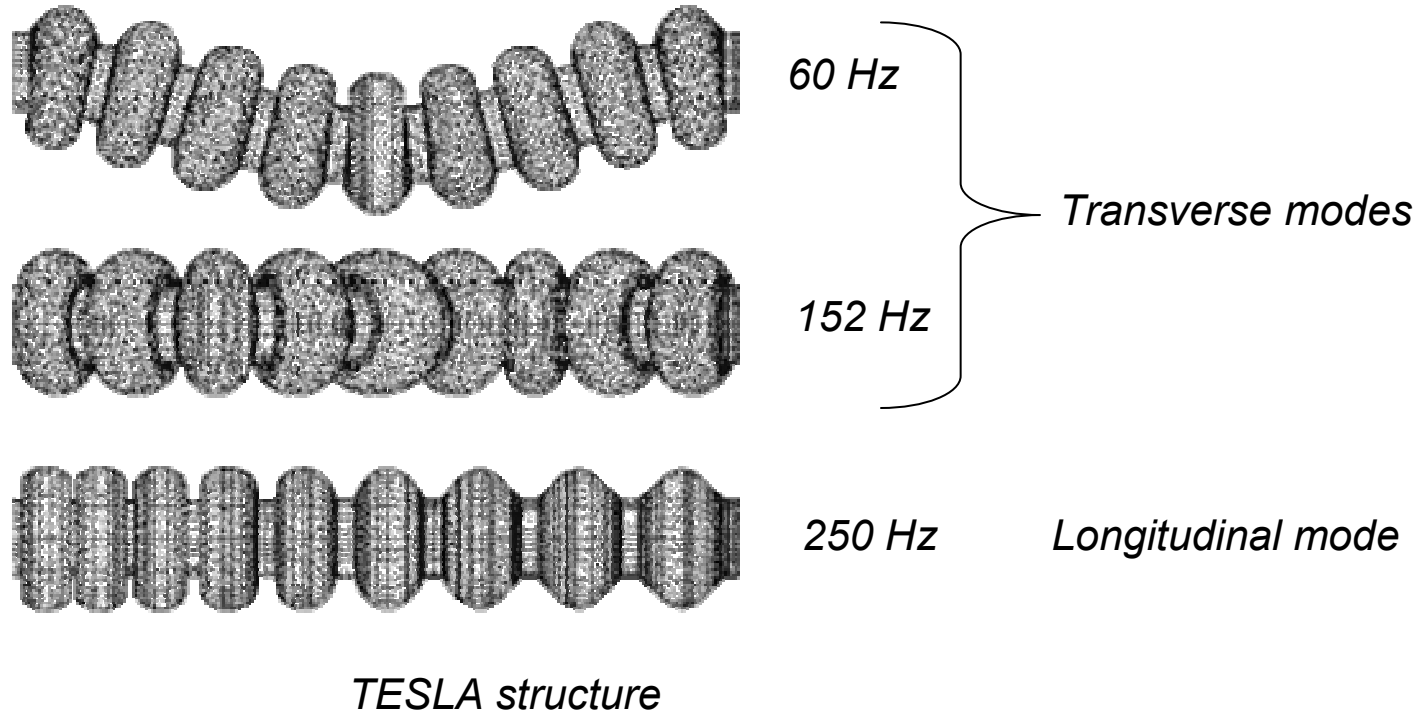
Essential for the operation of a pulsed accelerator

$$\Delta f = k_L (E_{acc})^2$$

$$k_L = -1 \text{ Hz}/(\text{MV}/\text{m})^2$$

Mechanical Design

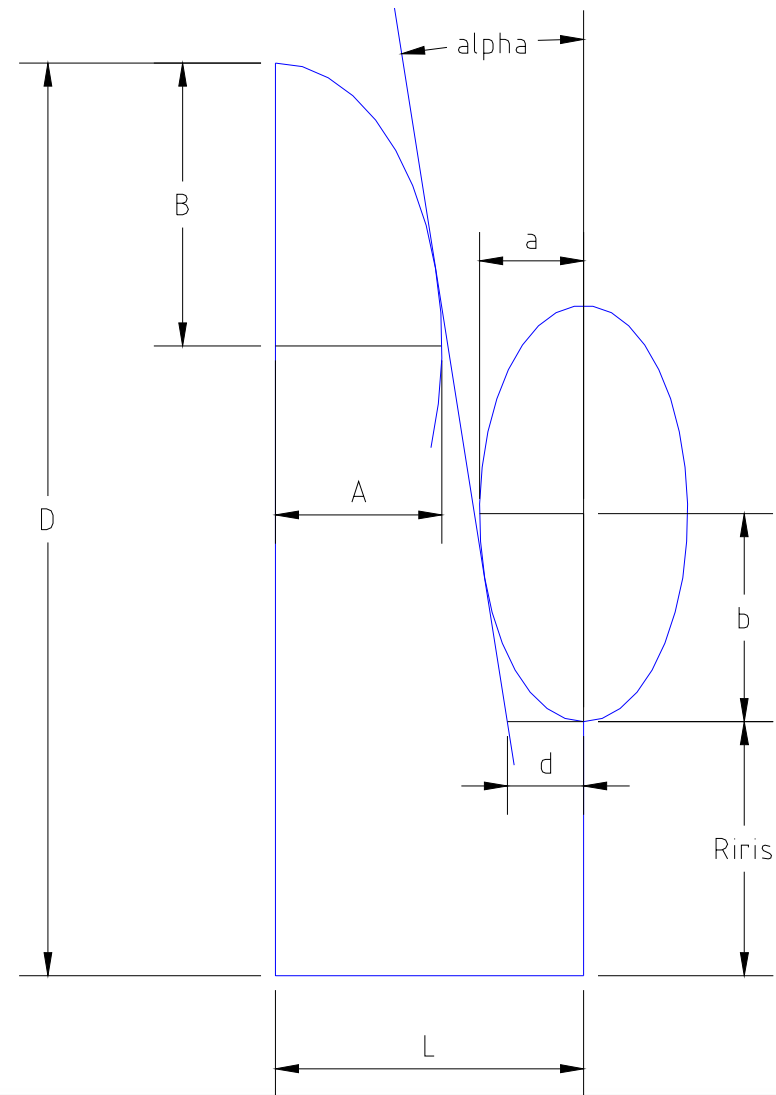
Mechanical Resonances of a multi-cell cavity



*The mechanical resonances modulate frequency of the accelerating mode.
Sources of their excitation: vacuum pumps, ground vibrations...*

SNS Cell Shape Parametrization

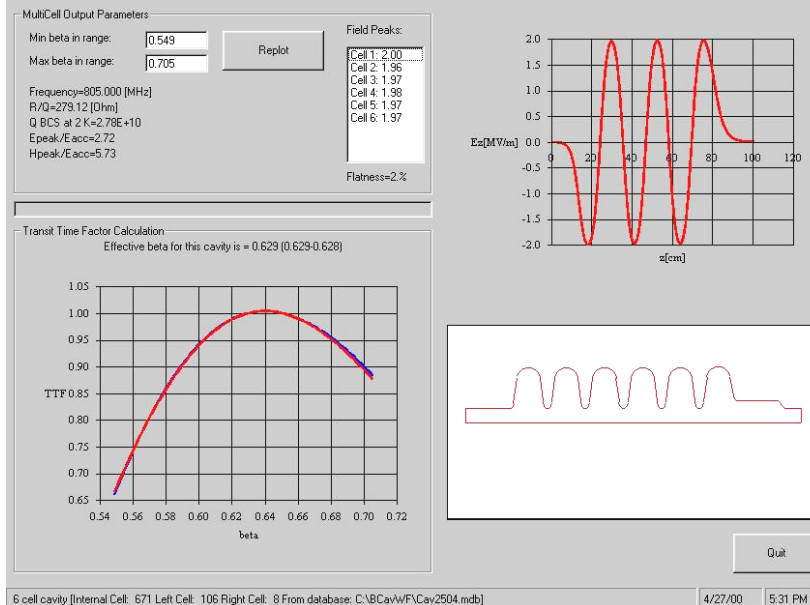
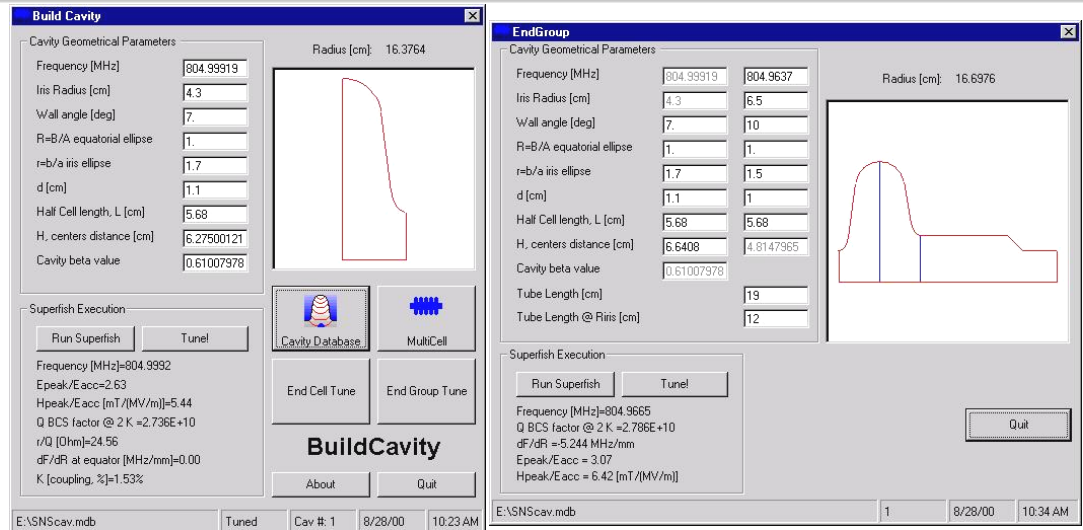
- Full parametric model of the cavity in terms of 7 meaningful geometrical parameters:
 - ✓ Ellipse ratio at the equator ($R=B/A$)
ruled by mechanics
 - ✓ Ellipse ratio at the iris ($r=b/a$)
Epeak
 - ✓ Side wall inclination (a) and position (d)
Epeak vs. Bpeak tradeoff and coupling k
 - ✓ Cavity iris radius Riris
coupling k
 - ✓ Cavity Length L
 β
 - ✓ Cavity radius D
used for frequency tuning
- Behavior of all e.m. and mechanical properties has been found as a function of the above parameters



Tools used for the parametrization

Using of the **parametric tool** developed at *INFN Milano* for the analysis of the cavity shape on the electromagnetic parameters:

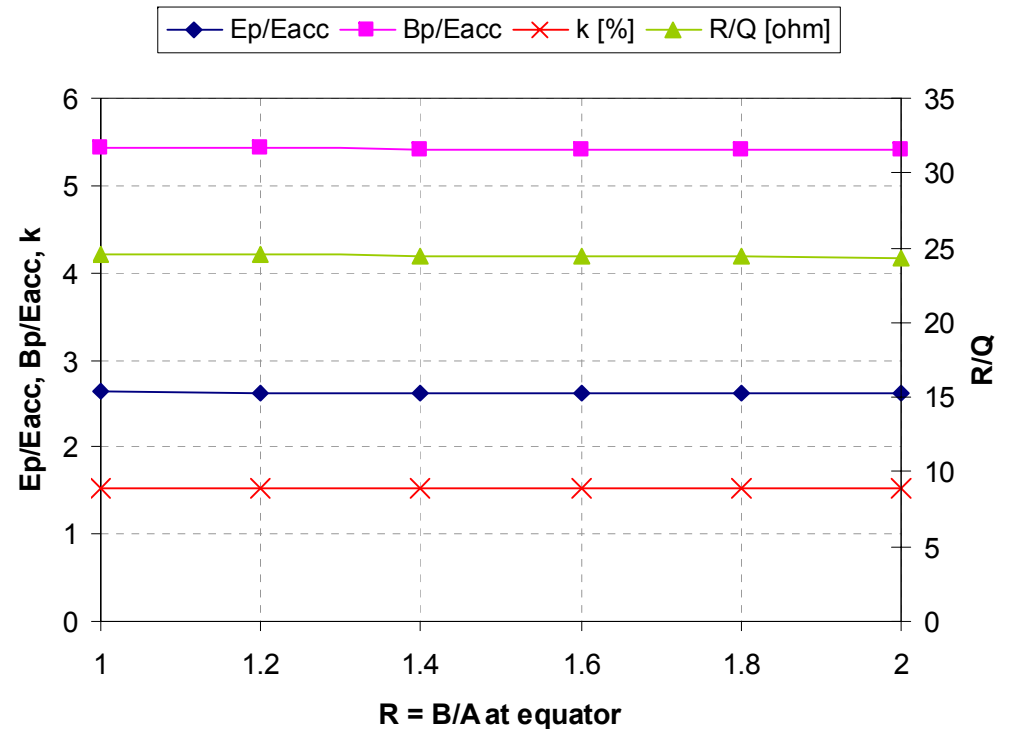
- All RF computations are handled by **SUPERFISH**
- **Inner cell tuning** is performed through the cell diameter, all the characteristic cell parameters stay constant: R , r , α , d , L , R_{iris}
- **End cell tuning** is performed through the wall angle inclination, α , or distance, d .
- **R , L and R_{iris} are independently settable.**
- **Multicell cavity** is then built to minimize the field unflatness, compute the effective β and the final cavity performances.
- A proper file to transfer the cavity geometry to **ANSYS** is then generated



Inner cell data
 $L = 56.8$ mm
 $R = 1$
 $r = 1.7$
 $\alpha = 7^\circ$
 $d = 11$ mm
 $R_{iris} = 43$ mm

R: “mechanical” parameter

- The equator aspect ratio (R) is a **free parameter** for what concerns the **π -mode e.m. design** but the cavity mechanical parameters are greatly affected by the equator shape:
- $R > 1$ allows better stress distribution in the unstiffened cavity but a **bigger Lorentz force coefficient**
- The cell tuning strategy doesn't affect the cell's performances (e.m. and mechanical)

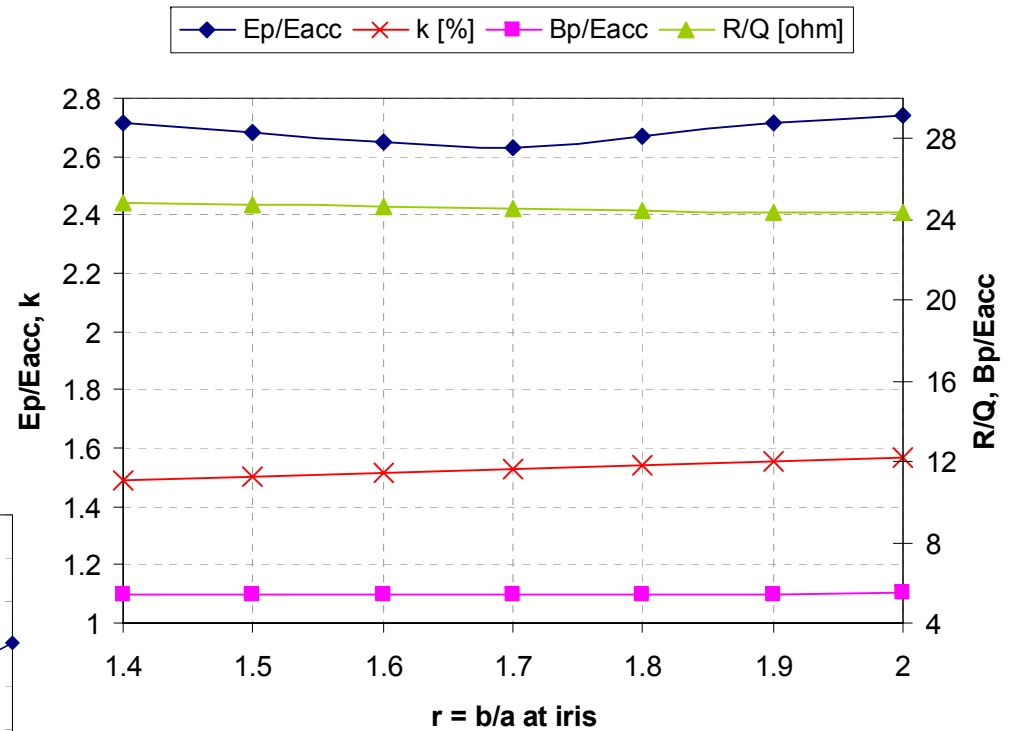
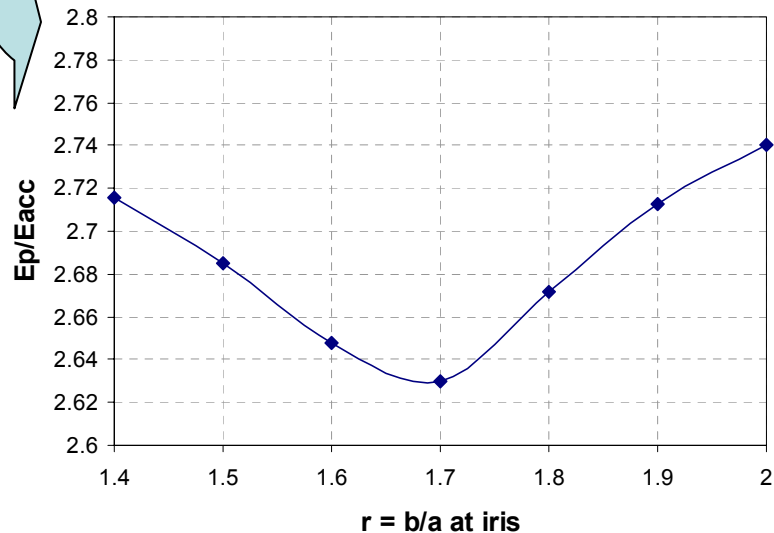
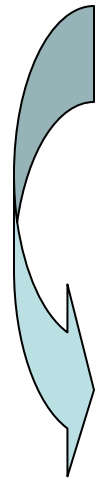


Reference data		
$L = 56.8$ mm	$r = 1.7$	$\alpha = 7^\circ$
$d = 11$ mm	$R_{iris} = 43$ mm	

Chosen $R = 1$

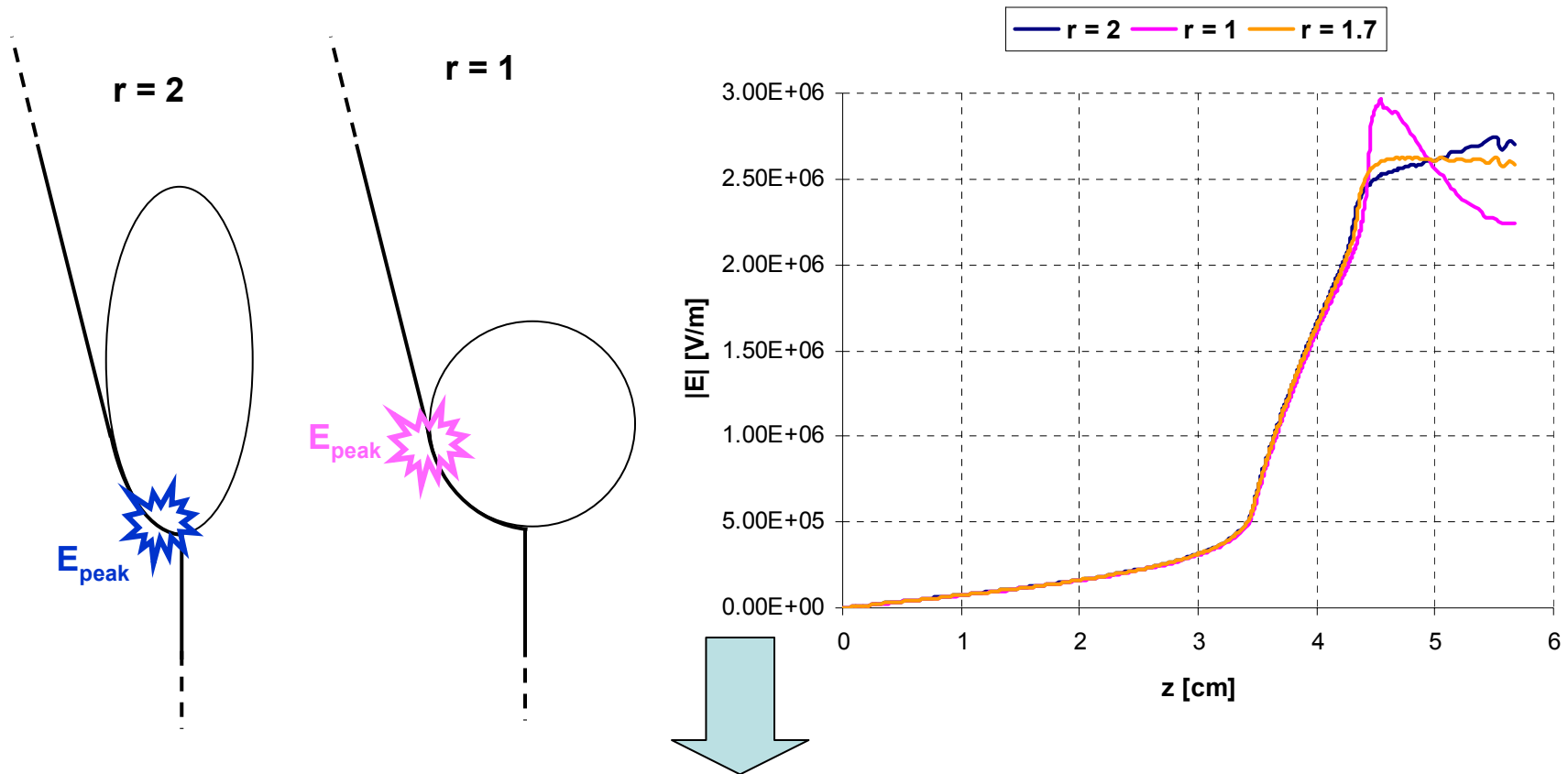
Optimal value for r

- The iris ellipse aspect ratio has always an **optimal** value that **minimize** the peak surface **E** field
- All the other cavity parameters (e.m. and mechanical) are unchanged



Reference data
 L = 56.8 mm R = 1 $\alpha = 7^\circ$
 d = 11 mm R_{iris} = 43 mm

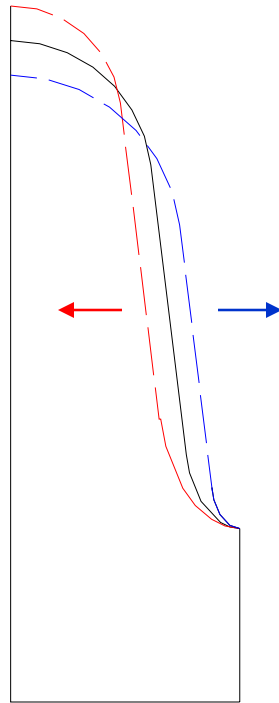
Optimal value for r



Optimization of r means **best** field distribution on the cell's nose

Chosen $r = 1.7$

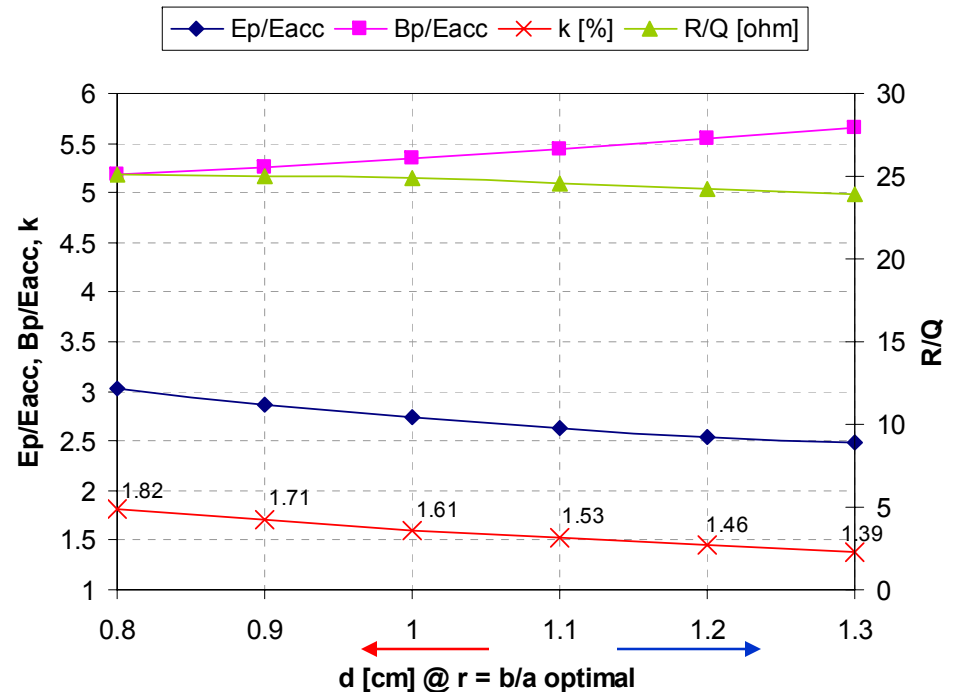
Influence of d @ constant R_{iris}



- d can be used to balance the **electric** and **magnetic** volumes of the cavity
 - The value of r is always optimal.
 - The **cell to cell coupling changes**, if R_{iris} is kept constant

Reference data
 $L = 56.8$ mm
 $R = 1$ $\alpha = 7^\circ$
 $r = \text{optimal}$
 $R_{iris} = 43$ mm

- Better mechanical performances are reached with decreasing d



Influence of d @ constant k

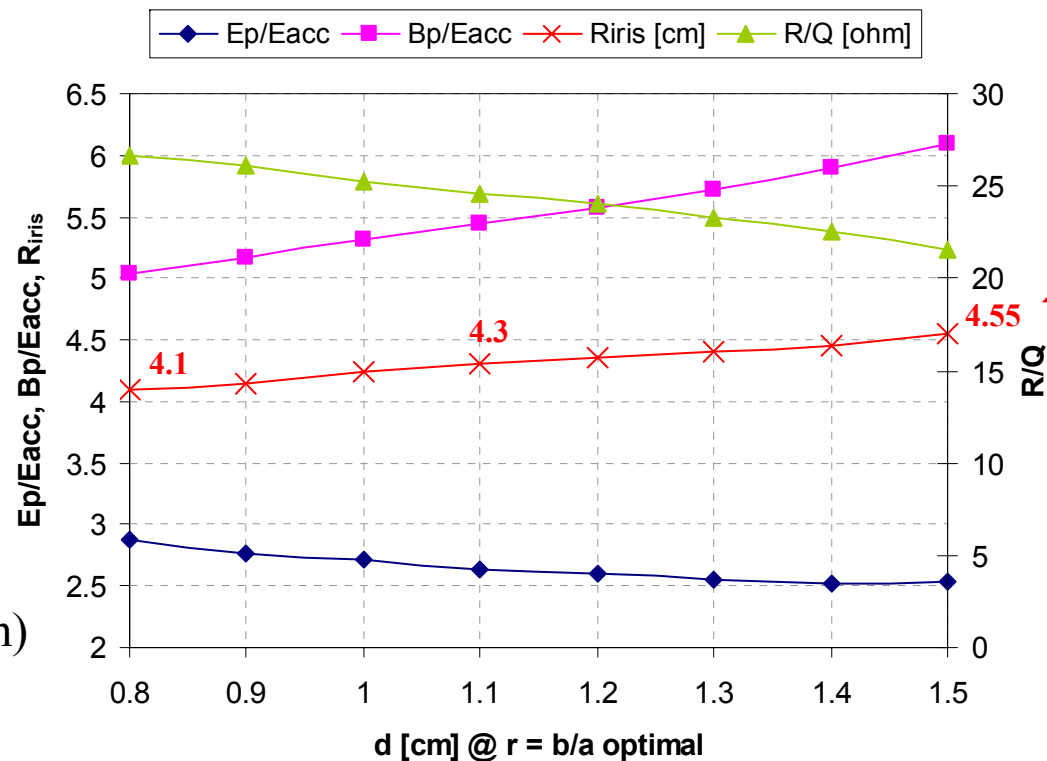
- If we want to keep a **constant cell to cell coupling** we have to **adjust Riris**

Reference data
 L = 56.8 mm
 R = 1
 $\alpha = 7^\circ$
 r = optimal
 k = 1.5 %

Chosen d=11 mm



$$B_{\text{peak}}/E_{\text{peak}} = 2.07 \text{ mT}/(\text{MV}/\text{m})$$

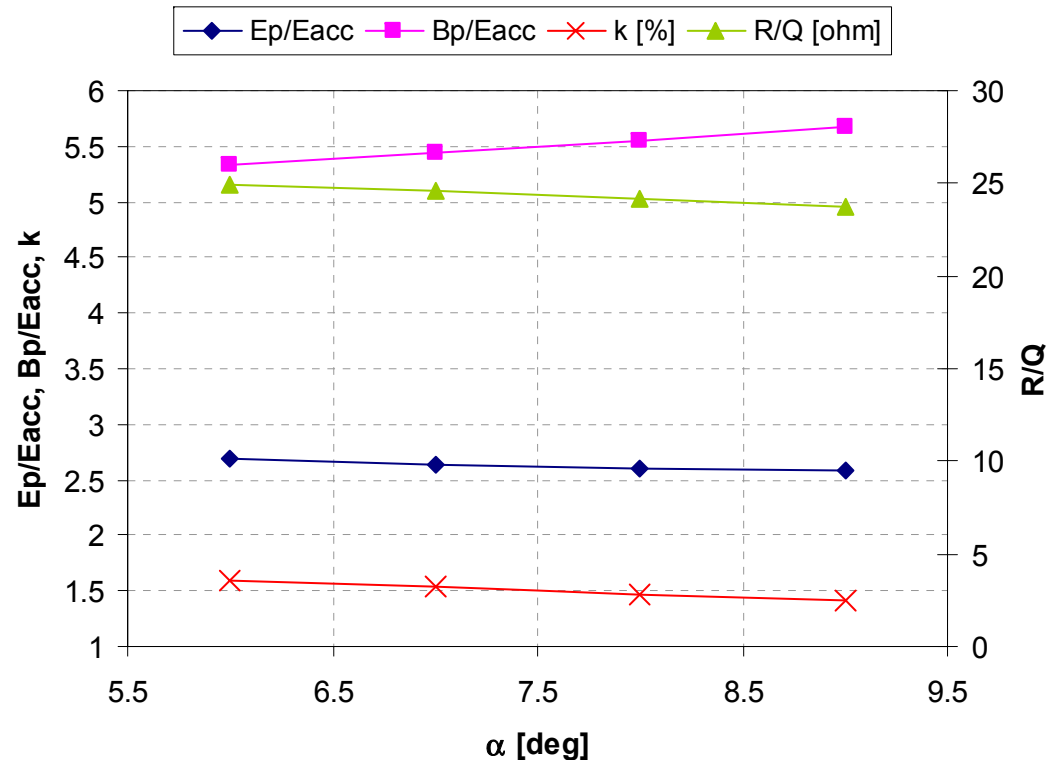


Dependence on α

The wall angle α slightly affects all the e.m. parameters, but has a **strong effect on the mechanical performances:**

- Lower values are preferred for **Lorentz force detuning**
- Too small α could be critical for **chemistry** and **cleaning**

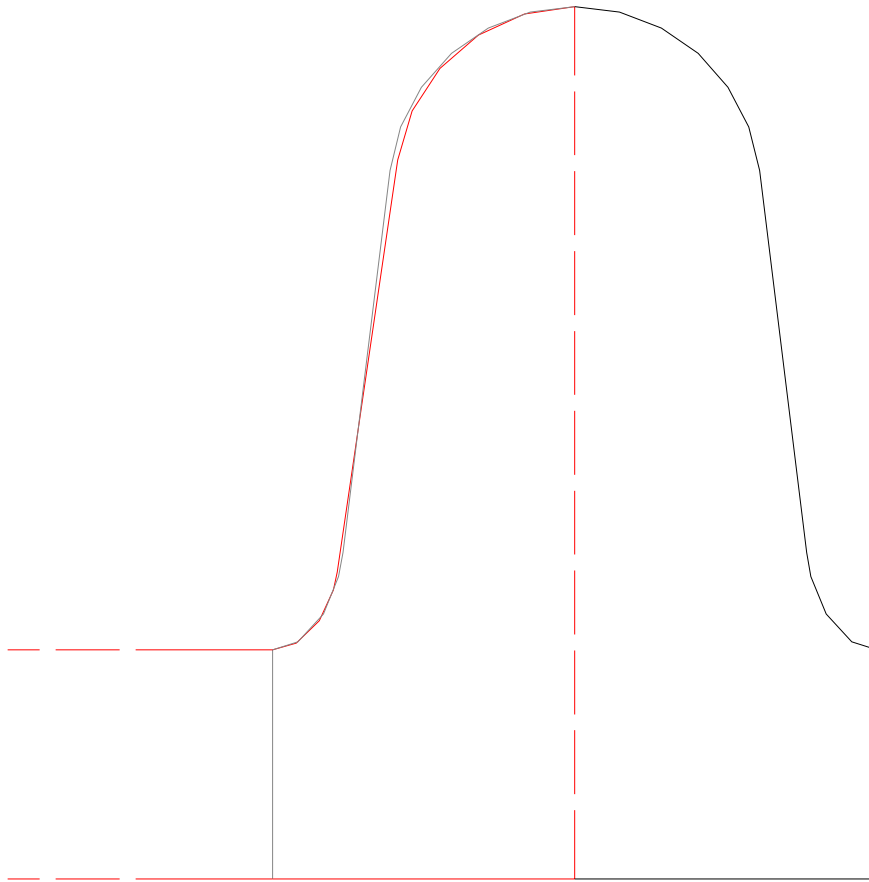
Chosen $\alpha = 7$ deg



Reference data

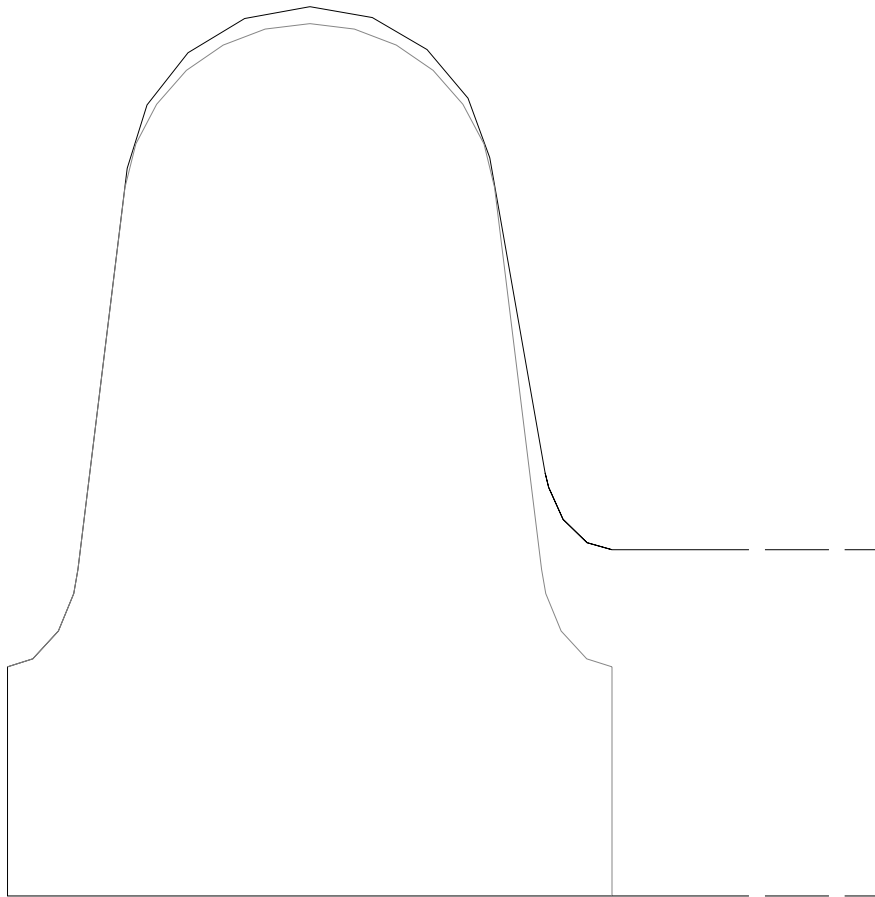
$L = 56.8$ mm $R = 1$
 $d = 11$ mm $r = 1.7$
 $R_{iris} = 43$ mm

End cell tune



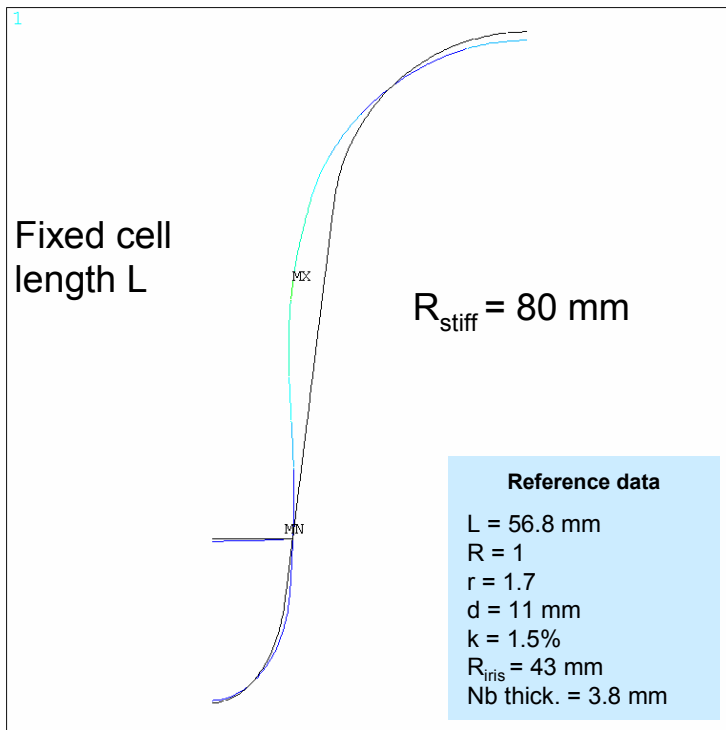
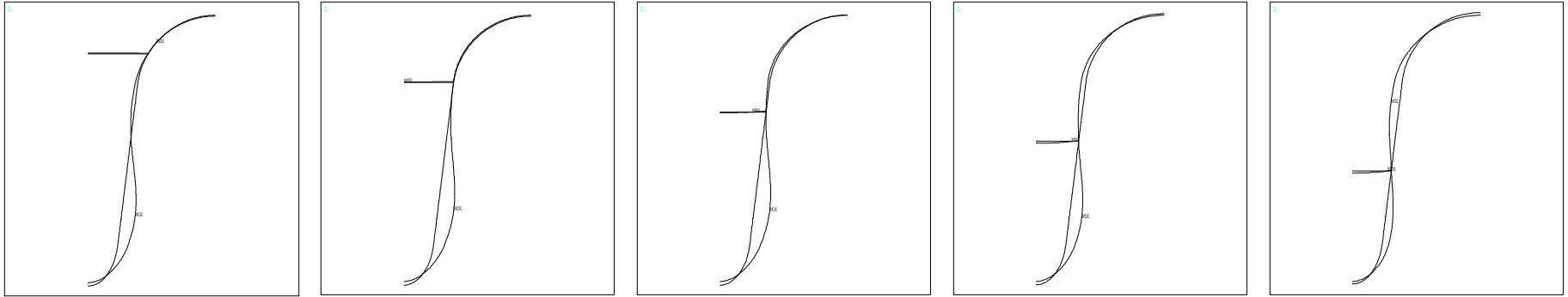
- d set 1 mm lower than the in-cell
- optimization of $r = b/a$ at iris
- Slater compensation (decrease of the magnetic volume) of the cut-off tube and d reduction ($\downarrow f$), increasing the wall angle α . This gives also the necessary stiffening to the end cell
- the frequency of end cell + tube is about 50 kHz lower than the in-cell's due to the asymmetry

End cell @ FPC side tune



- R_{iris} set to 65 mm to have enough field at the power coupler antenna
- d set 1 mm lower than the in-cell
- optimization of $r = b/a$ at iris
- α set to 10 deg to have the necessary stiffening
- Slater compensation (increase of the magnetic volume) of the cut-off tube ($\downarrow f$), d reduction ($\downarrow f$), α and R_{iris} increase ($\uparrow f$) by increasing the equator radius \Rightarrow 4 dies
- the frequency of end cell + tube is about 40 kHz lower than the in-cell's due to the asymmetry

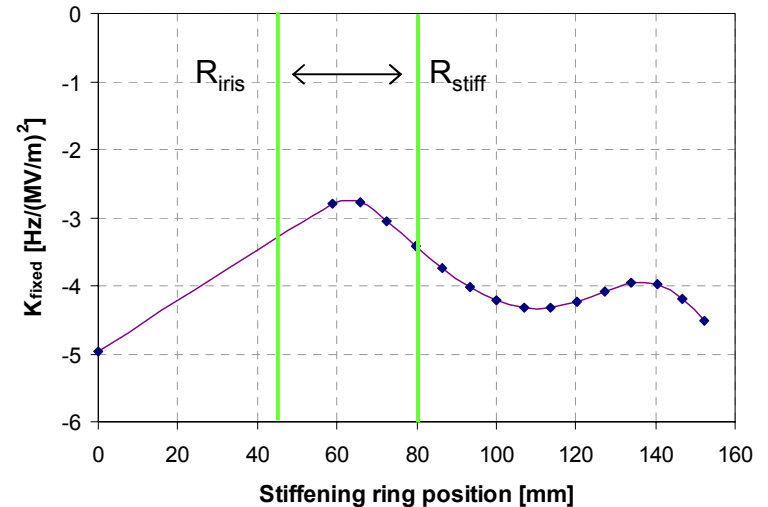
Optimal stiffening ring position



```

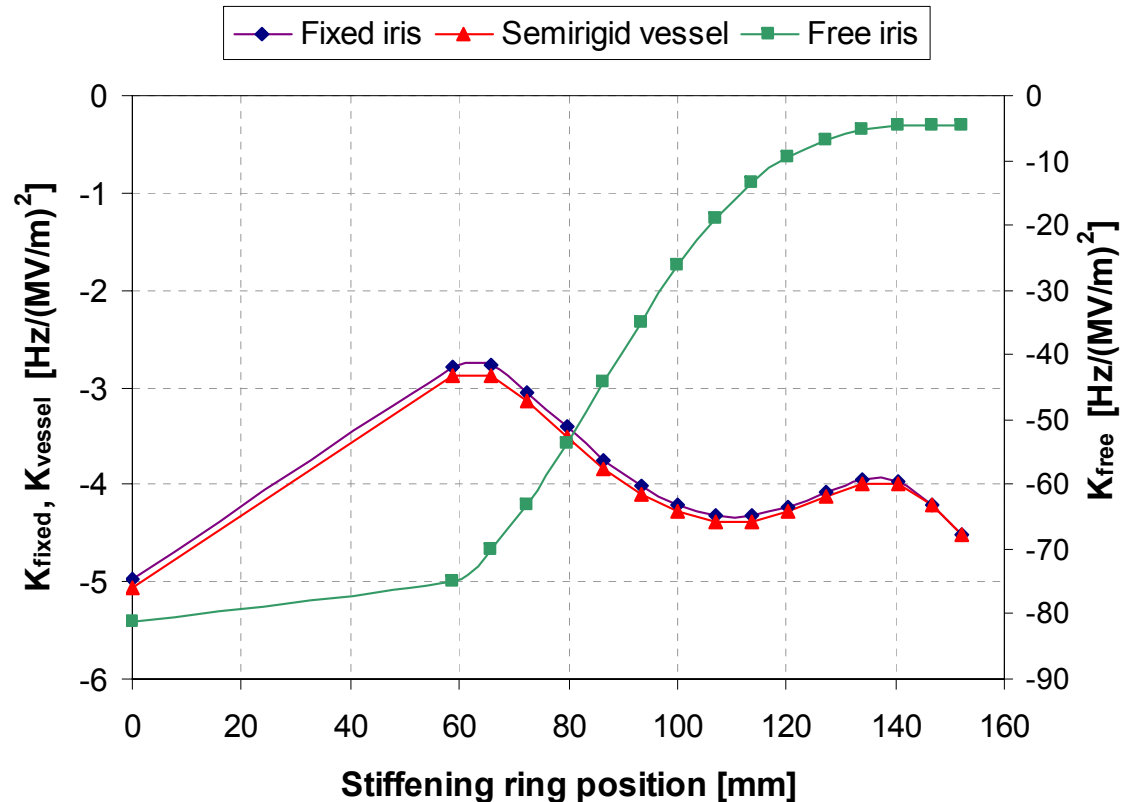
ANSYS 5.6
SEP 18 2000
15:27:02
PLOT NO. 13
NODAL SOLUTION
STEP=1
SUB =1
TIME=1
USUM      (AVG)
RSYS=0
PowerGraphics
EFACET=1
AVRES=Mat
DMX =.112E-05
SMN =.103E-07
SMX =.112E-05
0
.278E-06
.556E-06
.833E-06
.111E-05
.139E-05
.167E-05
.194E-05
.222E-05
.250E-05
    
```

The Lorentz forces coefficients for 15 different stiffening ring positions are evaluated automatically with ANSYS, preparing the geometry and reading the fields from the SFO output from SUPERFISH



K_L for different boundary conditions

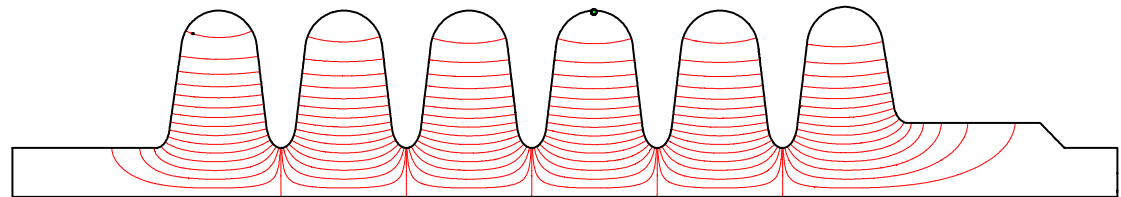
- The estimate for K_L strongly depends on the cell boundaries. We compute it for 3 different cases:
 - Fixed cell length
 - Free cell length
 - Helium Vessel/Tuning System (= 3 tubes with diameter 30 mm and thickness 2 mm)



$\beta_g = 0.61$ Cavity for SNS

Effective β that matches the TTF curve = 0.630

E_p/E_{acc}	2.72 (2.63 inner cell)
B_p/E_{acc} [mT/(MV/m)]	5.73 (5.44 inner cell)
R/Q [Ω]	279
G [Ω]	214
k [%]	1.53
Q_{BCS} @ 2 K [10^9]	27.8
Frequency [MHz]	805.000
Field Flatness [%]	2



KL70 = -2.9 [Hz/(MV/m)²] KL80 = -3.4 [Hz/(MV/m)²]

Nb thickness = 3.8 mm-

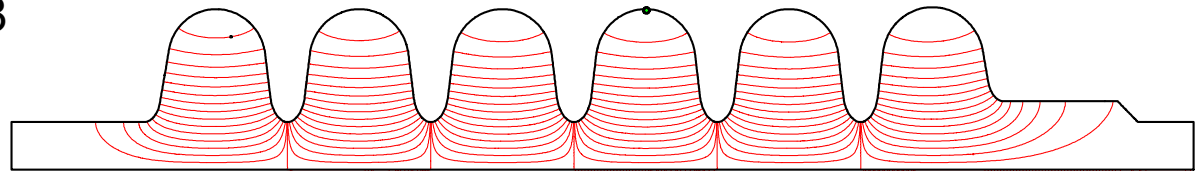
Geometrical Parameters

	Inner cell	End Cell Left	End Group (coupler)	
			Left	Right
L [cm]	5.68	5.68	5.68	
R_{iris} [cm]	4.3	4.3	4.3	6.5
D [cm]	16.376	16.376	16.698	
d [cm]	1.1	1.0	1.1	1.0
r	1.7	1.5	1.7	1.5
R	1.0	1.0		1.0
α [deg]	7.0	8.36	7.0	10.0

$\beta_g = 0.81$ Cavity for SNS

Effective β that matches the TTF curve = 0.832

E_p/E_{acc}	2.19 (2.14 inner cell)
B_p/E_{acc} [mT/(MV/m)]	4.72 (4.58 inner cell)
R/Q [Ω]	484.8
G [Ω]	233
k [%]	1.52
Q_{BCS} @ 2 K [10^9]	36.2
Frequency [MHz]	805.004
Field Flatness [%]	1.1



KL70 = -0.7 [Hz/(MV/m)²]

KL80 = -0.8 [Hz/(MV/m)²]

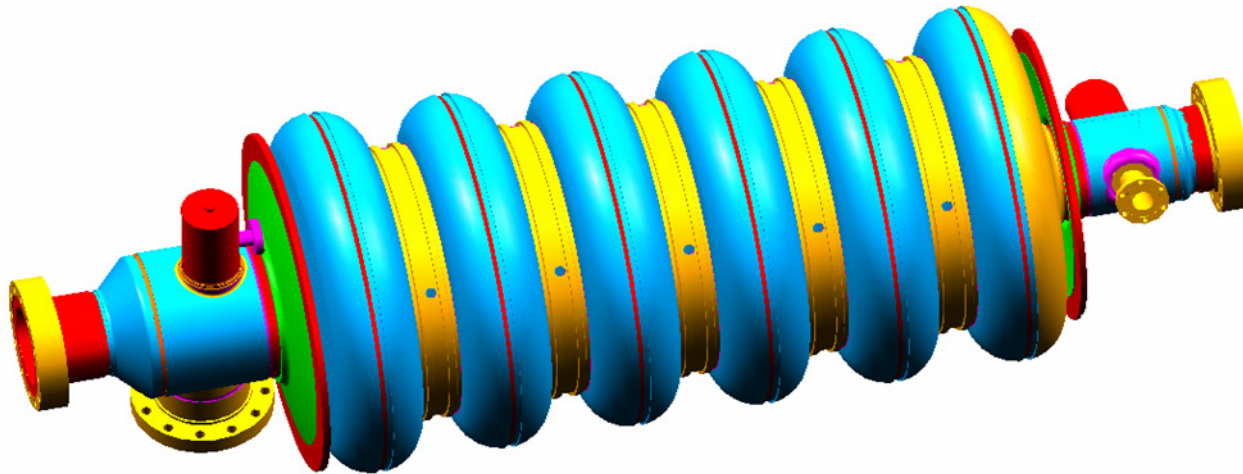
Nb thickness = 3.8 mm-

Geometrical Parameters

	Inner cell	End Cell Left	End Group (coupler)	
			Left	Right
L [cm]	7.55	7.55	7.55	
R_{iris} [cm]	4.88	4.88	4.88	7.0
D [cm]	16.415	16.415	16.611	
d [cm]	1.5	1.3	1.5	1.3
r	1.8	1.6	1.8	1.6
R	1.0	1.0		1.0
α [deg]	7.0	10.072	7.0	10.0

Stress and Modal Analysis

- Nominal Medium Beta Cavity



SNS Cavity Modal Analysis

Medium Beta Cavity

End Condition	Load (atm)	127 mm Rings (Hz)	70 mm Rings (Hz)	No Rings (Hz)
Fixed-Guided	-	85	48	38
Fixed-Fixed	-	126 (*204)	57 (*59)	48 (*42)
Fixed-Fixed Mid Supt	-	149 (*220)	95 (~*108)	88
Compressed 0.4mm	1.65	125	-	46
Compressed 1.25 mm	1.65	124	-	46

(*D. Schrage, LANL)

(~ Beta = 0.76)

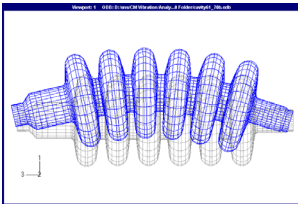
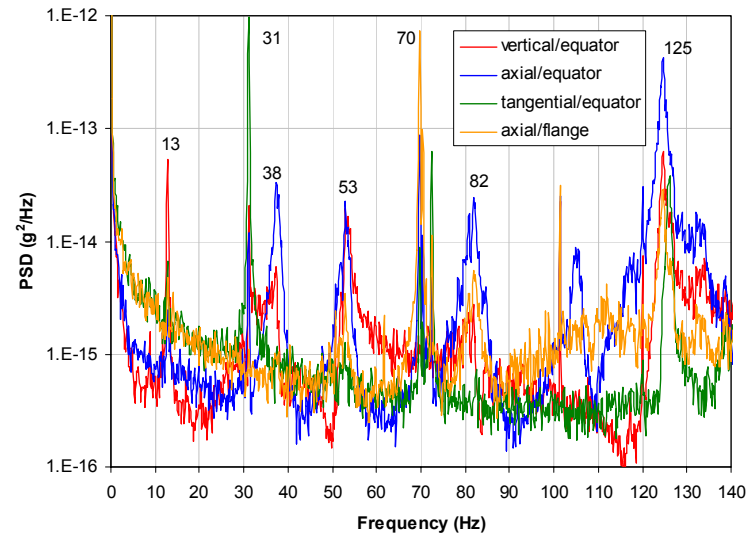
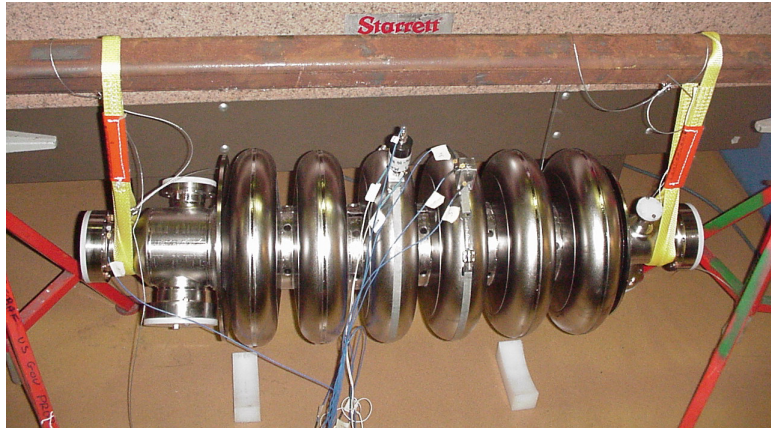
2000-0xxxx/vlb

SNS Cavity Modal Analysis

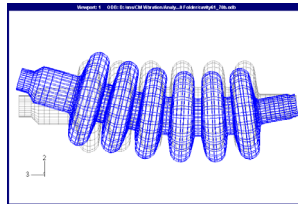
High Beta Cavity

End Condition	Load (atm)	127 mm Rings (Hz)	70 mm Rings (Hz)	No Rings (Hz)
Fixed-Fixed	-	120	-	46
Fixed-Guided	-	107	-	34
Compressed 0.4mm	1.65	120	-	44
Compressed 1.25 mm	1.65	119	-	44

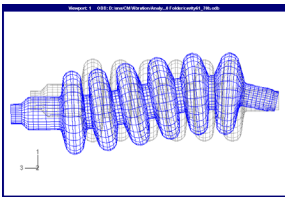
Mode Analysis, Beta = 0.81



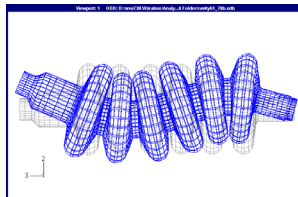
Mode 1 – 14 Hz



Mode 2 – 26 Hz



Mode 3 – 40 Hz



Mode 5 – 72 Hz

Mode	Natural Frequency (Hz)	
	Test Data	FE Analysis
1	13	14
2	31	26
3	38	40
4	53	48
5	70	72
6	82	83
7	125	124

SNS Cavity Mechanical Design Requirements

- Minimize/prevent microphonics
- Withstand loss of vacuum accident up to 5 atm
- Withstand cool down at 1.65 atm
- Adhere to intent of ASME B&P Code
 - Allowable Stress (S_m) = $2/3$ Yield Stress
 - Primary Membrane Stress (P_m) $\leq S_m$
 - P_m + Bending $\leq 1.5 * S_m$
 - P_m + Bending + Secondary Stress $\leq 3 * S_m$
 - Allowable Stresses

» Warm Niobium = 4,667 psi

» Cold Niobium = 53,333 psi

Medium Beta Stress Analysis

SNS Medium Beta Cavity Wall Stresses				
Compression (mm/end)	Loads (atm)	127 mm Stiffening Ring Max Stress (psi)	70 mm Stiffening Ring Max Stress (psi)	No Stiffening Ring Max Stress (psi)
0.2	1.65	-	-	-
0.4	1.65	3,960	-	4,310
0.5	1.65	4,610	-	4,550
0.75	1.65	7,500	-	4,670
1.25	1.65	17,500	5,730 (1.8 atm)	5,000
0.75	5	11,200	-	12,900
1.25	5	14,300	10,100	47,100

High Beta Stress Analysis

SNS High Beta Cavity Wall Stresses				
Compression (mm/end)	Loads (atm)	127 mm Stiffening Ring Max Stress (psi)	70 mm Stiffening Ring Max Stress (psi)	No Stiffening Ring Max Stress (psi)
0.2	1.65	3,040	-	-
0.4	1.65	6,350	-	3,140
0.5	1.65	8,070	-	3,350
0.75	1.65	12,500	-	3,940
1.25	1.65	21,400	-	5,830
0.75	5	11,500	-	9,130
1.25	5	14,300	-	9,590